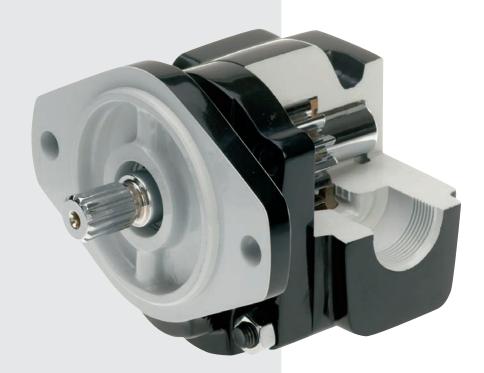




Service Manual PGP/PGM600

Effective:January 1, 2012Supersedes:All Others

PGP600 Series



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Single Pump

610/620/640 Service Manual

General Instructions

These service instructions will familiarize you with Parker's single and multiple pumps:

- their component parts
- the relative position of each part
- proper methods for assembly or disassembly of the units

To facilitate the repair of these units and before any work is done, we suggest that you first read all of the steps used in disassembly and assembly.

Dirt is the enemy of any hydraulic system. The first requirement for good maintenance of hydraulic equipment is cleanliness. MAKE SURE YOU DISASSEMBLE AND ASSEMBLE YOUR HYDRAULIC EQUIPMENT IN A CLEAN AREA.

The pictures show Model PGP620. Notes in the text cover variations between this unit and the other models.

It is important to airblast all parts and wipe them with a clean, lintless cloth before assembly.

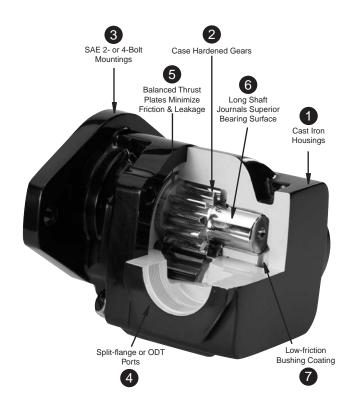
USE CAUTION IN GRIPPING ALL PARTS IN THE VISE TO AVOID DAMAGING MACHINED SURFACES.

A pump must be driven in the direction of rotation for which it was built; otherwise, pressure will blow the shaft seal. Check the exploded view and notes at right for proper direction of rotation.

Parker's Replacement Parts

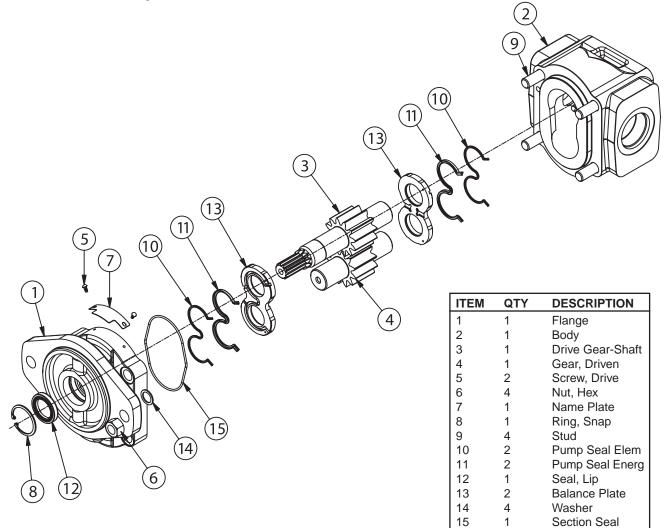
Parker's replacement parts are of original equipment standards. For assured quality of material and workmanship, and for compatibility in assembly, USE ONLY GENUINE PARTS.

Check all replacement parts before installing them to be certain that they were not damaged in shipment.

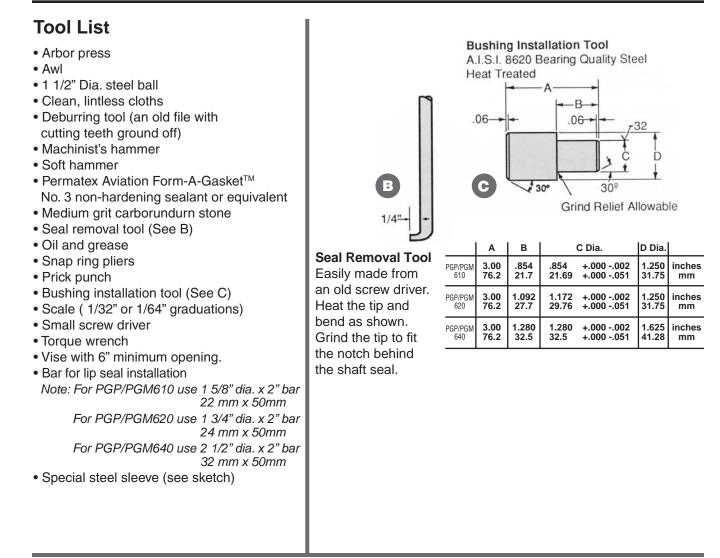




PGP/PGM600 Exploded View



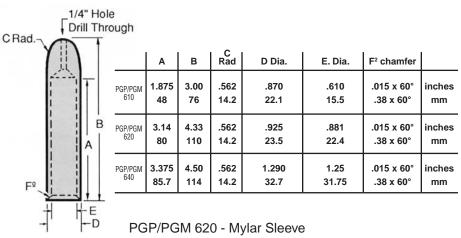




Special Steel Sleeve

The special steel sleeve is used to insert the drive shaft through the lip seal without damage and can be made from bar stock: For the PGP/PGM610 use a 1.00" (25.4 mm) dia. bar x 3.25" long (82.55 mm); for the PGP/PGM620 use a 1.375" (35 mm) dia. bar x 4.625" long (117 mm); for the PGP/ PGM640 use a 1.00" (25.4 mm) dia. bar x 4.625" long (117 mm). The drawing and chart give details for making this special tool.

NOTE: Please see tool list for lip seal bar installation.



PGP/PGM 620 - Mylar Sleev Part # 8682-133-00G



Start Disassembly Here

CAUTION:

- 1. If prying off sections becomes necessary, take extreme care not to mar or damage machined surfaces. Excessive force while prying can result in misalignment and seriously damage parts.
- 2. If parts are difficult to fit during assembly, tap gently with a soft hammer (never use an iron hammer).
- 3. Gears are closely matched, therefore they must be kept together as sets when removed from a unit. Handle with care to avoid damage to the journals or teeth. Avoid touching gear journals.
- 4. Never hammer bushings into bores; use an arbor press.



1) Place the pump in a vise with the drive shaft pointIng up. Caution: DO NOT GRIP ON OR NEAR ANY MACHINED SURFACES DURING ASSEMBLY OR DISASSEMBLY. Match-mark all sections. Be sure to align these marks when reassembling.

3) Lift off the flange. If prying is necessary, be careful not to damage the machined surfaces.

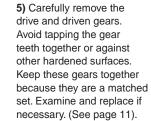


2) Use a wrench to remove the 4 cap screws on single units or the 4 hex nuts, studs and washers of multiple units.



4) Remove the balance plate. Examine and replace if necessary. See wear guide page 11.



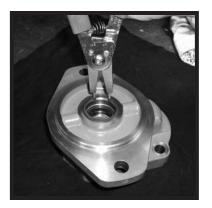




6) Carefully remove the balance plate from the bottom of body. Examine and replace if necessary. (See page 11)

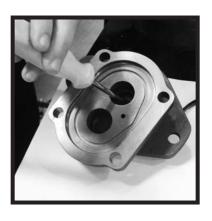


Catalog HY09-SM600/US Disassembly Information



7) Remove the snap ring (located in front of the shaft seal ring).

PGP/PGM600 Series Service Manual



8) Push out the shaft seal.



9) Inspect all bushings for scoring or discoloration and replace if necessary. Bushings cannot easily be removed with a puller. We suggest replacing with a new flange or body with bushings already installed.



10) Clean all faces of the body and mounting flange from sealant and dirt.



Start Assembly Here



1) If the bushings have been removed, deburr the bushing bores with an emery cloth. Rinse parts in a solvent. Air blast all the parts and wipe with a clean, lintless cloth before starting assembly.



2) Any bushings removed from the flange and body should be replaced and assembled in the drive bores with the groove to the top of unit (12 o'clock). Assemble the bushings in the driven bores with the groove to the bottom of the unit (6 o'clock).

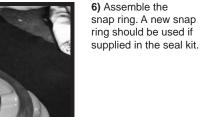


3) Bushings should be pressed into the bores, one at a time, using the special installation tool and an arbor press. Be sure the that the grooves (or seams) are positioned as stated in Step #2. The bushings must be pressed into the bores flush with the casting face. Be sure to support the castings so that they are square and level.

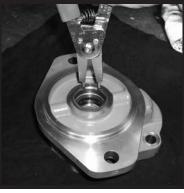
5) Press in the shaft seal to bottom of the shaft seal bore.



4) The shaft seal should be packed with some grease before installation in the bore shaft seals with metal OD need Loctite applied to outside of can.







8) Put a rope of Loctite 518 sealant out side the interlock track on the body (it's for corrosion



7) Check the section seal. Use of a new one is recommended in each case. Grease the new section seal to hold it in the groove.





protection)

Catalog HY09-SM600/US Assembly Information



9) Check balance plates and the sealing parts.

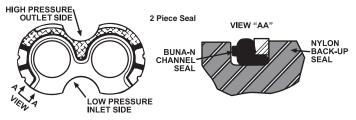


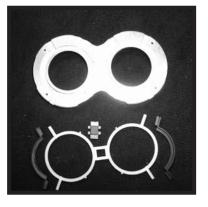
10) Assemble new sealing parts, the rubber seal first, the plastic seal on top. (Pump plate and seals.)

The instructions for proper installation for a 2 piece seal are as follows:

- 1. Turn the balance plate so the seal groove faces up
- 2. Place the soft black Buna-N seal into the seal groove with the flat side down
- 3. Place the hard white nylon back-up seal, flat side up, into the groove on top of the Buna-N seal

The proper seal installation is very important. If these seals are assembled upside down they will most likely fail in a short period of time under system pressure.





11) Motor balance plate and seals.



12) Assemble the lower balance plate into the body, sealing parts can be fixed with grease. Position is rotation sensitive!



13) Assemble the gears into the body, journals and other contact faces should be oiled with clean hydraulic fluid.



14) Put the upper balance plate on top of the gear set. Position is rotation sensitive!



15) Slide the steel shaft seal sleeve over the drive shaft. Lubricate the sleeve with grease to lubricate the shaft seal during assembly.



Catalog HY09-SM600/US Assembly Information



16) Fit the mounting flange careful from the top down to the body. Fit the interlocking track correctly.



17) Thread the fasteners into the body and tighten alternately or cross corner. Torque diagonally opposed fasteners to correct torque.

FASTENER TORQUE

PGP610/PGM610	43 lb-ft	58 Nm
PGP620/PGM620	70 lb-ft	95 Nm
PGP640/PGM640	103 lb-ft	140 Nm



Guideline for acceptable wear

Body

Wear in excess of .005" cut-out necessitates replacement of the body. Place a straight-edge across bore. If you can slip a .005 feeler gage under

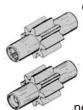


the straight-edge in the cut-out area, replace the body.

Pressure pushes the gears against the housing on the low pressure side. As the hubs and bushings wear, the cut-out

becomes more pronounced. Excessive cut-out in a short period of time indicates excessive pressure or oil contamination. If the relief valve settings are within prescribed limits, check for shock pressures or tampering.

When the cut-out is moderate, 005" or less, the body is in good condition and may be reused.



Gears

Any scoring on gear hubs necessitates replacement. Scoring, grooving, or burring of the outside diameter of the teeth requires replacement. Nicking, grooving, or fretting of teeth surfaces also necessitates replacement.

Drive Shafts



Replace if there is any wear detectable by touch in the seal area or at the drive coupling. The maximum allowable wear is .002".

Wear in the shaft seal area indicates oil contamination. Wear or damage to splines, keys, or keyways necessitates replacement.

Balance Plates



The balance plates seal the gear section at the sides of the gears. Wear here will allow internal slippage, that is, oil will bypass within the pump.

A maximum of .002" wear is allowable. Replace balance plates if they are scored, eroded or

pitted. Check center of balance plates where the gears mesh. Erosion here indicates oil contamination.

Pitted balance plates indicate cavitation or oil aeration. Discolored balance plates indicate overheating, likely due to insufficient oil.

Bushings



Bushings must be replaced if PTFE coating is worn through or if scored or blackened. Bushings should fit into the bore with a heavy press fit.

Seals and Gaskets



Replace all rubber and polymer seals, including all "0" rings, thrust plate channel seals, shaft seal and gasket seals.

🧹 Checks

Examine the checks, if unit has them, in the mounting flange to make sure that they are tight, and free of contamination.



Hydraulic Oil Recommendation

When choosing your Hydraulic Oil, duty cycle and oil temperature must be factored in to optimize your system performance Since hydraulic systems often work under extreme temperature changes, especially in moderate to severe duty cycles, the lubrication qualities of the oil in tight tolerance components is even more critical.

Viscosity choice is always a compromise; the fluid must be thin enough to flow easily but thick enough to seal and maintain a lubricating film between bearing and sealing surfaces. This film of oil helps to reduce friction and heat, which can ultimately lead to component damage.

Fluid temperature does affect viscosity. When choosing a fluid, it is important to consider the start-up and operating temperatures of the hydraulic system. In general, as the fluid warms, it gets thinner and its viscosity decreases. At the highest temperatures, the fluid must be thick enough to provide lubrication and minimize internal leakage. Low viscosity leads to the following problems:

- Higher leakage across all sealing gaps in the pump leading to lower volumetric efficiencies and heat.
- Heat will cause loss of lubrication and will create severe wear because of metal to metal contact, causing premature failure of the gears, thrust plates and bearings.

The opposite is true when the fluid cools, its viscosity increases. At the lowest temperatures, the fluid must be thin enough to flow readily. High viscosity oil leads to the following problems:

- Sealing and lubrication gaps not being filled, loss of lubrication
- Filling losses occur which causes cavitation damage to the pump

Under normal operating temperatures it is recommended to keep the temperature of the hydraulic fluid in the range of 120° F to 140° F (49° C to 60° C). Fluids may break down or oxidize at high temperatures, which leads to varnish or sludge deposits in the system and also reduces lubricity and results in reduced life of the unit. As a rule of thumb, operating temperatures over 176° F (80° C) reduce the service life by half for every 50° F (10° C) temperature increase, and should be avoided.

Petroleum Oils (Mineral-Based)

Viscosity Recommendations

- Optimum operating viscosity is considered to be about 100 SUS (20 cSt).
- Minimum approximately 50 60 SUS (7.5 10 cSt) Maximum at start up: approximately 7500 SUS (1600 cSt)

Recommended Viscosity Grades

	Viscosity at	Viscosity at
Grade	100°F (40°C)	210°F (100°C)
ISO 32	165 SUS (32 cSt)	44 SUS (5 cSt)
ISO 46	240 SUS (46 cSt)	49 SUS (7 cSt)
SAE 10	150 SUS (32 cSt)	41 SUS (4 cSt)
SAE 20	300 SUS (71 cSt)	51 SUS (7 cSt)

Other Desirable Properties

Viscosity Index: 90 minimum Aniline Point: 175 minimum

Additives Usually Recommended

Rust and Oxidation (R & O) Inhibitors Foam Depressant

Note: Antiwear (AW) additives are not necessarily recommended. In some instances the presence of zinc compounds can actually be harmful to copper, bronze, or brass components used in the system. The use of AW oil is optional with our gear units.

General Recommendations

High quality hydraulic oils are essential for satisfactory performance and long life of any hydraulic system. Such oils are usually prepared from highly refined, turbine oil stocks with which select additives are compounded. We suggest following the manufacturer's specifications or the recommendations of a reputable oil supplier for the specific oil requirements on your machine.

A high viscosity oil will generally give better performance and life than a thin oil. Oil of around 100 SUS (20 cSt) will give optimum performance. Your selection should be as near to optimum as possible at operating temperature but not so heavy at start-up as to cause cavitation. Cold start-up procedures which allow the use of heavier oils should prove worthwhile by increasing pump life. The oil must be clean and contain less than 0.1% water.

Operating Temperature

The optimum oil operating temperature is in the range of 120°-140°F (49°-60°C). Oil operating temperature should not exceed 200°F (93°C) with a maximum of 180°F (82°C) generally recommended. If the oil temperature will be above 180°F (82°C) for significant periods of time, then Viton (FKM) seals should be used. High temperatures may result in rapid oil deterioration and may point out the need for an oil cooler or a larger reservoir. The nearer to optimum temperature, the longer the service life will be of the oil, pump and other components.

Cold Weather Operation

Oils for use in cold weather should have a viscosity not exceeding 7500 SUS (1620 cSt) at the minimum start up temperature and a pour point of at least 20°F (7° C) below that temperature. Experience on the Alaskan North Slope has been satisfactory without using special oils or fluids. Start-up procedures must allow for a gradual warm-up and equipment should not be operated at full pressure until the oil reaches a reasonably fluid state.

Inlet Vacuum

Vacuum measured at the inlet port of the pump generally should not exceed 5 in. (13 cm) Hg. Higher vacuum can result in cavitation which may severely damage the pump. A usually acceptable rule of thumb is that the inlet line velocity should not exceed 8 fps (2.5 m/s). A long inlet line or the use of several fittings may necessitate increasing the line size. We suggest that each inlet port of a tandem pump have its own line from the reservoir.

Reservoir

Reservoir capacity in gallons should at least equal total pump output in GPM When filling the reservoir, oil should pass through a 100-mesh screen. Pour only clean oil from clean containers into the reservoir. The reservoir should have a breather to allow air in or out. The filler cap and breather should be sealed to prevent moisture from entering. A hydraulic oil water content of as little as 0.1% can cause damage to hydraulic components

Filtration

Good filtration assures improved service life at today's high operating pressures. System filtration is recommended that will maintain a contamination level according to ISO 4406: 21/19/16 for 2000 psi (140 bar), 19/17/14 for 3000 psi (210 bar) and 17/15/12 for 4000 psi (275 bar) The specific filter recommendation should come from your equipment manufacturer or filter supplier

A 100 mesh screen should generally be used in the suction line leading to the pump. It should be of sufficient size to handle twice the pump capacity. The screen must be cleaned and checked regularly to avoid pump and system damage

Oil and filters should be changed on a regular schedule and the system flushed in accordance with the original equipment manufacturer's recommendations. Reservoir air breather filters should be cleaned periodically

Filtration is not a substitute for practicing cleanliness and proper preventive maintenance



Water Base Fire Resistant Fluids

Two types of water base fluids (WBF} may be used with our gear pumps and motors.

Both types of WBF come in various viscosity grades. Select the grade best suited to the equipment and its operation in terms of pressure, speed, temperature, duty cycle, etc. The fluid used should be recommended by the O.E.M. or a reputable fluid supplier

Operations outside the range of 400 to 2000 rpm and over 3000 psi (2000/2500 psi in the wider gear widths) should be reviewed with your Parker sales representative

Water-in Oil (60/40) invert Emulsions

Invert emulsions are approved for use with our bushing style pumps at pressures up to 3000 psi or 500 psi below catalog rated pressures, whichever is lower

With roller bearing pumps, life may be reduced to 20% to 50% of that experienced with petroleum oil. Reducing the pressure and/or gear width can extend pump life appreciably.

Water Glycol Solutions

Water glycol solutions of the types normally used in hydrostatic systems are recommended for use with our bushing style pumps. These consist of about 60% glycol and about 40% water with additives to improve lubricity and other characteristics. Pressures up to 3000 psi are approved, depending on the gear width. Water glycol solutions are not approved for use with our roller bearing pumps.

WBF Filtration

Filtration that seems to give the best results consists of a 100-mesh inlet screen and a return line filter. For water base fluids, the inlet screen should be sized up three to four times the pump capacity. The return line filter should be of a rating and size recommended by the fluid and filter manufacturers to achieve a recommended ISO contamination level

Note: Finer filtration may be required by other components in the system

High Water Base Fluids (HWBF)

The use of 95/5 emulsion is not recommended

Phosphate Ester

Phosphate ester does not appear to effect pump performance and service life, but Viton (FKM) seals should be used with this fluid. Viscosity characteristics of phosphate ester fluid limit the recommended ranges of operating and ambient temperatures. Questions on the use of fluids with our equipment should be discussed with a sales representative or Product Support Dept.

Comments On The Use Of Other Oils And Fluids

Biodegradable Oil (Vegetable-Based)

Oils of this type with properties similar to recommended petroleum oils may be used with our bushing style pumps only. These are not approved for use in our roller bearing pumps. Performance, pressure ratings and durability are not adversely affected in bushing style pumps.

Automatic Transmission Fluid (ATF)

In general these oils have low viscosity and may be used only at reduced operating pressures and oil temperatures

Diesel Fuel, Kerosene, Coal Oil

Although sometimes used as a dilutant for cold weather operations, their use is not recommended because they are insufficiently refined products

Transformer Oil

Sometimes used for extremely cold weather operation. It is not generally recommended as it becomes too thin at normal operating temperatures. Oil to U.S. Military Spec MIL-H-5606 is in this category.

OPERATING LIMITS GENERALLY RECOMMENDED WITH VARIOUS FLUIDS

Fluid:	Max.Operating Temp.	Max. Inlet Line Velocity	Max. Inlet Vacuum
Petroleum Oil	180°F(82°C)	8 fps(2.5m/s)	5" (13cm) Hg
WIO Emulsion	150°F(65°C)	4 fps (1.2m/s)	0" (0cm) Hg
Water Glycol	150°F(65°C)	4 fps (1.2m/s)	0" (0cm) Hg

Note: These figures represent generally accepted maximums and will not prove satisfactory in all installations. For very severe duty cycles, it will likely be advantageous to design and operate the system at something less than these maximum limits

- DO NOT USE ANY TYPE OF FLUID NOT RECOMMENDED IN THIS BULLETIN WITHOUT FIRST CONSULTING OUR PRODUCT SUPPORT DEPT
- OBTAIN YOUR FINAL FLUID RECOMMENDATION FROM YOUR FLUID SUPPLIER



Recommended Start-up Procedure For New or Rebuilt Pump

Before installing a new or rebuilt pump, back off the main relief valve until the spring tension on the adjusting screw is relieved. This will avoid the possibility of immediate damage to the replacement unit in the event that the relief valve setting had been increased beyond the recommended operating pressure, prior to removing the old unit.

Before connecting any lines to the pump, fill all ports with clean oil to provide initial lubrication. This is particularly important if the unit is located above the oil reservoir.

After connecting the lines and mounting the replacement unit, operate the pump at least two minutes at no load and at low rpm (400 min.) During this break-in period, the unit should run free and not develop an excessive amount of heat. If the unit operates properly, speed and pressure can then be increased to normal operating settings.

Reset the main relief valve to its proper setting while the pump is running at maximum operating engine (motor) speed for the vehicle.

> ALWAYS USE AN ACCURATE GAUGE WHEN ADJUSTING THE RELIEF VALVE PRESSURE SETTING.



Recommended Test Procedure

Make certain that there is an adequate supply of oil for the pump; at least one gallon of oil for each gpm of pump capacity.

If one section of a tandem pump is being tested, make sure that all other sections not being tested are adequately supplied with oil. If any of the other sections run dry, or if plugs are left in ports, serious and permanent damage will result.

The oil should be a good quality hydraulic oil rated at 150 SSU at 100° F., with the oil temperature held at 120° F. plus or minus 50° F. (Test procedures are described in detail in SAE handbooks; see Hydraulic Power Pump Test Procedure, SAE J745c.)

The feed line must be of adequate size with no more than 5" mercury vacuum adjacent to the pump inlet. As a rule, the feed line must provide a feed flow velocity not in excess of 8 feet per second.

Feeding hot oil into a cold pump may cause the pump to seize. Jog the pump by momentarily starting and stopping repeatedly the driving engine or motor to gradually equalize pump and oil temperatures.

Run the pump at least two minutes at no load and moderate speed (not over 1500 rpm). If the pump becomes excessively hot, shut down immediately and locate the problem source.

Gradually increase pressure on pump, in 500 psi increments until the desired test pressure has been reached. This should take about five minutes.

Delivery should run close to rated catalog performance figures, which are averaged from testing several pumps. A 5% lower reading may be used as a rated minimum if new or relatively new parts have been used. When rebuilding the pump with parts from the original pump, which, while worn, appear satisfactory for reuse, a 10% or 15% lower reading may be permitted, depending on the performance expected from the equipment. One's own experience will prove the best guide here. Many repairmen measure the output at normal operating speed and at zero pressure, then again at 1000 psi (or the operating pressure of the equipment) and allow a volume decrease approximating the listing below. It is a suggested reference only which makes allowance for reused parts.

At test speeds other than 1800 rpm, gpm delivery will vary almost proportionately, but the same (drop-off) figures should be used.

Be sure to run the pump in the direction for which it was designed and built. Driving the pump in the wrong direction will build up pressure behind the shaft seal, damaging it and necessitating replacement.

After completing testing procedures, the pump is ready for installation and immediate duty operation on equipment. Again, it must be remembered that to prevent seizure, hot oil must not be fed into a cold pump.

GPM	DELIVERY
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at 1800 rpm	GPM DROP OFF AT			
100 psi	1000 psi/70 bar	2000 psi/140 bar	3000 psi/210 bar	
10-30	11/2-3	2-31/2	21/2-4	
30-50	2-3	21/2-4	3-41/2	
50-70	21/2-31/2	3-5	31/2-51/2	





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2. Payment: Payment shall be made by Buyer net 30 days from the date of delivery of the items purchased hereunder. Amounts not timely paid shall bear interest at the maximum rate permitted by law for each month or portion thereof that the Buyer is late in making payment. Any claims by Buyer for omissions or shortages in a shipment shall be waived unless Seller receives notice thereof within 30 days after Buyer's receipt of the shipment.

3. Delivery: Unless otherwise provided on the face hereof, delivery shall be made F.O.B. Seller's plant. Regardless of the method of delivery, however, risk of loss shall pass to Buyer upon Seller's delivery to a carrier. Any delivery dates shown are approximate only and Seller shall have no liability for any delays in delivery.

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by Buyer. Unless otherwise agreed, Seller shall have the right to alter, discard or otherwise dispose of any special tooling or other property in its sole discretion at any time.

8. Buyer's Property: Any designs, tools, patterns, materials, drawings, confidential information or equipment furnished by Buyer or any other items which become Buyer's property, may be considered obsolete and may be destroyed by Seller after two (2) consecutive years have elapsed without Buyer placing an order for the items which are manufactured using such property, Seller shall not be responsible for any loss or damage to such property while it is in Seller's possession or control.

9. Taxes: Unless otherwise indicated on the face hereof, all prices and charges are exclusive of excise, sales, use, property, occupational or like taxes which may be imposed by any taxing authority upon the manufacture, sale or delivery of the items sold hereunder. If any such taxes must be paid by Seller or if Seller is liable for the collection of such tax, the amount thereof shall be in addition to the amounts for the items sold. Buyer agrees to pay all such taxes or to reimburse Seller therefore upon receipt of its invoice. If Buyer claims exemption from any sales, use or other tax imposed by any taxing authority, Buyer shall save Seller harmless from and against any such tax, together with any interest or penalties thereon which may be assessed if the items are held to be taxable.

10. Indemnity For Infringement of Intellectual Property Rights: Seller shall have no liability for infringement of any patents, trademarks, copyrights, trade dress, trade secrets or similar rights except as provided in this Part 10. Seller will defend and indemnify Buyer against allegations of infringement of U.S. Patents, U.S. Trademarks, copyrights, trade dress and trade secrets (hereinafter 'Intellectual Property Rights'). Seller will defend at its expense and will pay the cost of any settlement or damages awarded in an action brought against Buyer based on an allegation that an item sold pursuant to this contract infringes the Intellectual Property Rights of a third party. Seller's obligation to defend and indemnify Buyer is contingent on Buyer notifying Seller within ten (10) days after Buyer becomes aware of such allegations of infringement, and Seller having sole control over the defense of any allegations or actions including all negotiations for settlement or compromise. If an item sold hereunder is subject to a claim that it infringes the Intellectual Property Rights of a third party, Seller may, at its sole expense and option, procure for Buyer the right to continue using said item, replace or modify said item so as to make it noninfringing, or offer to accept return of said item and return the purchase price less a reasonable allowance for depreciation. Notwithstanding the foregoing, Seller shall have no liability for claims of infringement based on information provided by Buyer, or directed to items delivered hereunder for which the designs are specified in whole or part by Buyer, or infringements resulting from the modification, combination or use in a system of any item sold hereunder. The foregoing provisions of this Part 10 shall constitute Seller's sole and exclusive liability and Buyer's sole and exclusive remedy for infringement of Intellectual Property Rights.

If a claim is based on information provided by Buyer or if the design for an item delivered hereunder is specified in whole or in part by Buyer, Buyer shall defend and indemnify Seller for all costs, expenses or judgments resulting from any claim that such item infringes any patent, trademark, copyright, trade dress, trade secret or any similar right.

11. Force Majeure: Seller does not assume the risk of and shall not be liable for delay or failure to perform any of Seller's obligations by reason of circumstances beyond the reasonable control of Seller (hereinafter 'Events of Force Majeure'). Events of Force Majeure shall include without limitation, accidents, acts of God, strikes or labor disputes, acts, laws, rules or regulations of any government or government agency, fires, floods, delays or failures in delivery of carriers or suppliers, shortages of materials and any other cause beyond Seller's control.

12. Entire Agreement/Governing Law: The terms and conditions set forth herein, together with any amendments, modifications and any different terms or conditions expressly accepted by Seller in writing, shall constitute the entire Agreement concerning the items sold, and there are no oral or other representations or agreements which pertain thereto. This Agreement shall be governed in all respects by the law of the State of Ohio. No actions arising out of the sale of the items sold hereunder or this Agreement may be brought by either party more than two (2) years after the cause of action accrues.

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About Parker Hannifin Corporation

Parker Hannifin is a leading global motion-control company dedicated to delivering premier customer service. A Fortune 500 corporation listed on the New York Stock Exchange (PH), our components and systems comprise over 1,400 product lines that control motion in some 1,000 industrial and aerospace markets. Parker is the only manufacturer to offer its customers a choice of hydraulic, pneumatic, and electromechanical motion-control solutions. Our

Company has the largest distribution network in its field, with over 7,500 distributors serving more than 350,000 customers worldwide.

Parker's Charter

To be a leading worldwide manufacturer of components and systems for the builders and users of durable goods. More specifically, we will design, market and manufacture products controlling motion, flow and pressure. We will achieve profitable growth through premier customer service.

Product Information

North American customers seeking product information, the location of a nearby distributor, or repair services will receive prompt attention by calling the Parker Product Information Center at our toll-free number: 1-800-C-PARKER (1-800-272-7537). In the UK, a similar service is available by calling 0500-103-203.



The Climate & Industrial Controls Group

designs, manufactures and markets system-control and fluid-handling components and systems to refrigeration, air-conditioning and industrial customers worldwide.

The Fluid Connectors Group designs, manufactures and markets rigid and flexible connectors, and associated products used in pneumatic and

The Aerospace Group

servicing of control systems

and related high-technology

markets, while achieving

growth through premier

customer service.

fluid systems.

is a leader in the development, design, manufacture and

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The Seal Group designs, manufactures and distributes industrial and commercial sealing devices and related products by providing superior quality and total customer satisfaction.

The Hydraulics Group designs, produces and markets a full spectrum of hydraulic components and systems to builders and users of industrial and mobile machinery and equipment.





The Filtration Group designs, manufactures and markets quality filtration and clarification products, providing customers with the best value, quality, technical support, and global availability.

The Automation Group is a leading supplier of pneumatic and electromechanical components and systems to automation customers worldwide.





The Instrumentation Group is a global leader in the design, manufacture and distribution of highquality critical flow components for worldwide process instrumentation, ultra-high-purity, medical and analytical applications.



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